Understanding and solving validation errors

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Validation errors may arise during design with NextGen. These types of errors are part of the normal work cycle with the software: the purpose of this article is to provide the user with the tools to understand and solve them independently.

The validation process

When the program has all the data required by the user to calculate a component, it proceeds with its validation. Validation is a two-step check that aims to provide the user with a series of intermediate and final values, as well as a result summarized by the traffic light icon.

Step 1: geometric validation

The first validation step is independent of the calculation code adopted and includes a geometric validation. The purpose of this validation is to understand whether the component set can be realistically represented in the 3D workspace and whether it is possible to perform the subsequent check on it.

Cylindrical shell "Shell"				🔯 Auto size 👝 🖻 🔀
😭 Essentials	General		🛃 Helper image [🧕	Preliminary report 🛛 💼 Usage factor
🚰 General	Name / Position	Shell		
Design conditions	Material	SA-516 70 - Plate Database Edit		
🛱 Geometry	Undertolerance method	Absolute value V	<u> </u>	
8 Ligaments	Undertolerance	3 🔦 mm		
∫x External loads	Geometry			
📣 Weight	Standard pipe	Pipes database		
Reporting	Length	L 1000 mm		
	Thickness	T 1 mm		
	Inside diameter	D 500 mm		
	Outside diameter	Do 502 mm		
			+	U U
😢 Errors (1) 🔥 Warnings	(0) 🚯 Messages (0) 📾 Relationships (0)			🔍 Expand
Property D	Pescription			Actual
Thickness N	Net thickness is less or equal to zero: increase the gross t	hickness or decrease corrosion and undertolerance.		-2.00 mm
		Save for lat	ter 📑 Load 🔞 Help	2 OK Cancel

Example of a cylinder with a geometric validation error

For example, creating a cylinder with a nominal thickness of 1 mm, having set a tolerance of 3 mm, generates a geometric error that indicates to the user that the net thickness value is negative. To resolve the error, it is necessary to increase the thickness.

Geometric validation errors prevent the component design window from being closed by clicking OK, since the 3D component cannot be physically added to the workspace

Step 2: code validation

Once the first geometric validation step has been passed, the program proceeds with the application of the rules of the calculation code in use. Following the previous example, once the thickness has been increased to obtain a positive net thickness, NextGen applies the rules of ASME VIII Div. 1 and produces the following result:

€ External loads	Geometry				
Weight	Standard pipe	Pipes database			
Reporting	Length	L 1000 💌 mm			
E Reporting	Thickness	T 4 💌 mm	Do Di		· · · · - <mark>·</mark>
	Inside diameter	D 500 🔹 mm			
	Outside diameter	Do 508 💌 mm			
					A
			(т)	٣
			<u> </u>		
S Errors (1) 🗥 Warnings	(0) 🚯 Messages (0) 📾 Relationships (0)				🔍 Expand
Property [Description		Required	Actual	Reference
Thickness	nternal pressure: The minimum thickness of shells and heads used in comp naterials listed in Table UCS-23, shall be 3/32 in. (2.5 mm)	vressed air service, steam service, and water service, made from) exclusive of any corrosion allowance	5.50 mm	4.00 mm	UG-16(b)

Example of code evaluation of a cylinder

Here, according to UG-16(b), a steam service has been set for this calculation. NextGen detects an insufficient minimum thickness and reports it to the user.

This type of error allows the window to be closed by clicking on the OK button. It is therefore possible to keep the component in question in error status, in case you want to resolve it at a later time.

The validation table

By carefully observing the validation table, you can obtain important information for resolving errors.

Property	Description	Required	Actual	Reference
<u>Thickness</u>	Internal pressure: The minimum thickness of shells and heads used in compressed air service, steam service, and water service, made from materials listed in Table UCS-23, shall be 3/32 in. (25 mm) exclusive of any corrosion allowance	5.50 mm	4.00 mm	UG-16(b)

- **Property:** indicates which property the error refers to. The property name is clickable, so you can position yourself directly on the value to be modified to change the calculation
- **Description**: description of the error which, where possible, reports the exact wording present in the calculation codes
- Required: the value required by the calculation to ensure that it is valid
- Actual: the value currently calculated, based on the data available to the program
- Reference: chapter, paragraph, formula of the regulation whose application is generating the error

Not all the data reported here are always available. In some cases, the program cannot calculate a required value, or in the case of geometric validation there is no reference to a norm paragraph.

Calculation error analysis

The most common case of validation error concerns, depending on the type of component, an

insufficient thickness or a resistance lower than required.

😰 Cylindrical shell "Shell"						
😭 Essentials	General			🛃 Helper image 🛛 🛕	Preliminary report	📕 Usage factor
🚰 General	Name / Position	Shell		Preliminary report:	Cylindrical she	ell 🔨
Design conditions	Material	SA-516 55 - Plate Data	abase Edit	According to: Asme VIII	Div. 1 Ed. 2023 U	G-27, UG-28
Geometry	Undertolerance method	Absolute value	~ < > 🔞	Internal pressure		
	Undertolerance	3 🔹 mm		Allowable stress	s	108.00 MPa
& External loads	Geometry			Allowable stress at roc	om c	T 109 00 MPa
Maight	Standard pipe	Pipes database		temperature	5	100.00 101 0
• Weight	Length	L 1000 🜩 mm		Geometry		
Reporting	Thiskness	T 6 mm		Outside diameter	E	00 512.00 mm
	Thickness			Adopted thickness	t	6.00 mm
	Inside diameter	D 500 🖶 mm		Corrosion allowance	с	0 mm
	Outside diameter	Do 512 🔹 mm		Wall undertolerance	с	3.00 mm
				Joint efficiency	E	0.70
				Internal pressure		
				Minimum required thic	kness tr	6.37 mm
				Minimum required thic	kness	
				according to UG-16(b)	(4), tr UG-1	6(b) 5.50 mm
				considering corrosion		t > tr: Ko
						~
😵 Errors (1) 🛕 Warnings (0) 🕦 Messages (0) 📾 Relationships (0)						
Property D	escription				Required	Actual
Thickness II	nternal pressure: Current thickness is lower than minimum required thickness fo	or internal pressure			6.37 mm	6.00 mm

Cylinder with insufficient internal pressure thickness

The case under consideration, a carbon steel cylinder with a thickness of 6 mm insufficient to withstand a pressure of 1 MPa at 180°C, would lead as a first choice to increase the nominal thickness by 1 mm, so as to satisfy the verification.

However, observing and analyzing the calculation report can help to understand what happens and above all find alternative solutions.

We recommend experimenting with this type of approach with components whose calculation is simple and direct, such as a cylinder, and then moving on to more complex scenarios such as flanges or openings.

Required thickness for circumferential stress, UG-27(c)(1)	$\begin{split} t_r &= \frac{P(R+c+c')}{S \cdot E' - 0.6 \cdot P} + c + c_e + c' \\ t_r &= \frac{1.00(250.00+0+3.00)}{108.00 \cdot 0.70000 - 0.6 \cdot 1.00} + 0 + 0 + 3.00 \end{split}$	=	6.37 mm	0.251 in
Required thickness for longitudinal stress, UG-27(c)(2)	$\begin{split} t_r &= \frac{P(R+c+c')}{2\cdot S\cdot E''+0.4\cdot P} + c + c_e + c' \\ t_r &= \frac{1.00(250.00+0+3.00)}{2\cdot 108.00\cdot 0.70000+0.4\cdot 1.00} + 0 + 0 + 3.00 \end{split}$	=	4.67 mm	0.184 in
Minimum required thickness	$t_r = \max[t_{r,circ}, t_{r,long} \ t_r = \max[6.37, 4.67]$,] =	6.37 mm	0.251 in
Item service	Service	=		Steam
Minimum required thickness according t corrosion	to UG-16(b)(4), considering tr UG-16(b)	=	5.50 mm	0.217 in
			t ≥ tr (6.00 mm	≥ 6.37 mm): Ko

Calculation report of the cylinder with insufficient thickness at internal pressure

NextGen highlights the validation problem in red. From the reported formula, it can be seen that the minimum thickness is the highest value between two formulas. Having identified the formula that is determining this value, it can therefore also be analyzed.

Required thickness for circumferential stress, UG-27(c)(1)	$t_r = \frac{P(R+c+c')}{S \cdot E' - 0.6 \cdot P} + c + c_e + c'$ $t_r = \frac{1.00(250.00 + 0 + 3.00)}{108.00 \cdot 0.70000 - 0.6 \cdot 1.00} + 0 + 0 + 3.00$:	6.37 mm
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The correspondence between the input values and the final output is starting to become clearer. For example, the radius R in the numerator indicates a value that, if increased, will have the effect of increasing the minimum thickness required. The pressure P intervenes in both the numerator and denominator, although to a lesser extent.

Above all, the welding efficiency is in the denominator, as is the allowable: this means that <mark>increasing</mark> <mark>these values will result in a lower minimum required thickness</mark>.

It is now evident that, in the comparison $tr \ge t$, the designer can not only increase t to satisfy the calculation, but also intervene to reduce tr.

Required thickness for circumferential stress, UG-27(c)(1)	$\begin{split} t_r &= \frac{P(R+c+c')}{S\cdot E' - 0.6\cdot P} + c \\ t_r &= \frac{1.00(250.00+0+1)}{138.00\cdot 0.70000-0} \end{split}$	$+ c_e + c'$ $- \frac{3.00}{0.6 \cdot 1.00} + 0 + 0 + 3.00$	=	5.64 mm	0.222 in
Required thickness for longitudinal stress, UG-27(c)(2)	$\begin{split} t_r &= \frac{P(R+c+c')}{2\cdot S\cdot E''+0.4\cdot P} + c\\ t_r &= \frac{1.00(250.00+0+3)}{2\cdot 138.00\cdot 0.70000+0} \end{split}$	$+ c_e + c'$ 3.00) $0.4 \cdot 1.00 + 0 + 0 + 3.00$	=	4.31 mm	0.170 in
Minimum required thickness		$t_r = \max[t_{r,circ}, t_{r,long}]$ $t_r = \max[5.64, 4.31]$	=	5.64 mm	0.222 in
Item service		Service	=		Steam
Minimum required thickness according t corrosion	o UG-16(b)(4), considering	tr UG-16(b)	=	5.50 mm	0.217 in
				t≥tr(6.00mm t≥	tr UG-16(b): Ok

Cylinder verification satisfied

Moving from a SA 516-55 material to SA 516-70, the allowable is increased from 108 MPa to 138 MPa, satisfying the verification.

Complex error resolution (NaN, infinity)

The procedure illustrated in the previous paragraph becomes essential when dealing with more complex errors. For example, it may happen that NextGen gives a result with the value Not a Number (NaN) or infinity $\pm \infty$.

This means that the calculation involves one or more invalid mathematical steps, such as:

- square root of a negative number
- division by zero

You can already tell when this happens from the design window:

😢 Errors (3) 🛕 Warnings (0) 🚯 Messages (1) 📾 Relationships (0)					
Description	Required	Actual			
Internal pressure: Current thickness is lower than minimum required thickness for internal pressure 18.00 mm					

Example of infinite requested value

S Errors (8) 🔬 Warnings (0+2) 🚯 Messages (0) 📾 Relationships (0)		🔍 Expand
Description	Required	Actual
Load combination Erection: Section 1-1 is not verified	223.33 MPa	NaN MPa

Example of current value Not a Number

Repeating the previous procedure, we produce the calculation report and identify where the problem is.

Head thickness		
Joint efficiency	Z =	1.00000
Head thickness	$s_0 = \frac{p_1 \cdot r}{1.2 \cdot f \cdot z} + c + c_e + t$ $s_0 = \frac{1.03 \cdot 616.00}{1.2 \cdot 0 \cdot 1.00000} + 1.60 + 1.60 + 2.00$	<mark>∞ mm</mark>
	s≥s0 (18	.00 mm ≥ ∞ mm): Ko

Division by zero in calculating the minimum thickness of a bottom

The check that fails must be identified, its formula and finally the value in the formula that leads to an invalid result. In this case, it is clear that it is a division by zero and that zero is the admissible value of the bottom in use.

To resolve the error, it is therefore sufficient to intervene on the material, taking care to choose one for which it is possible to calculate the admissible value or, alternatively, specifying an admissible value manually.

In the second case, the report highlights that NaN is the value assigned to the *Weight Factor* for the *Lifting lug*:

Lifting lug - Lifting Lugs						
	Metric Units					
Design temperature		т =	20.00 °C	68.00 °F		
Material:	SA/EN 10028-2 P355GH - Plate (≤60)					
Length		L =	150.00 mm	5.906 in		
Thickness		s =	6.00 mm	0.236 in		
Offset from shell border		=	685.00 mm	26.969 in		
Angular offset				0 °		
Weightfactor		=		NaN		
Shock factor		f =		1.40000		
Radius		R =	40.00 mm	1.575 in		

In this case, it is a value automatically calculated by the program; if the program does not have all the elements to calculate this value, it can produce an invalid value as in this case. Again, the user can unlock the input box and enter a manual value, so as to complete the calculation correctly.

Final considerations

In most cases, when a validation error is encountered, the procedure adopted by NextGen can be considered correct; the calculation report is the tool to adopt to understand where the failed validation comes from and therefore which input values to modify to obtain a satisfactory verification.

In rare cases, the program may fail to adopt the rules of the calculation code: if you think you have encountered this situation, you can open a support request via Ask for Support, along with a detailed numerical calculation, for example in the form of an annotated PDF or Excel sheet, which clearly shows the expected values, possibly with references to the formulas of the calculation code or the standard used and where these are calculated incorrectly by NextGen.